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PTO/SB/05 (4/98)
Approved for use through 09/30/2000. OMB 0651-0032

Patent and Trademark Office: U.S. DEPARTMENT OF COMMERCE

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**UTILITY
PATENT APPLICATION
TRANSMITTAL**

(Only for new nonprovisional applications under 37 C.F.R. § 1.53(b))

Attorney Docket No.	AIR COMP
First Inventor or Application Identifier	Donald C. Erickson
Title	AIR COMPRESSION IMPROVEMENT
Express Mail Label No.	EE451582748US

APPLICATION ELEMENTS

See MPEP chapter 600 concerning utility patent application contents.

1. <input checked="" type="checkbox"/> * Fee Transmittal Form (e.g., PTO/SB/17) (Submit an original and a duplicate for fee processing)	5. <input type="checkbox"/> Microfiche Computer Program (Appendix)
2. <input checked="" type="checkbox"/> Specification [Total Pages <input type="text" value="9"/>] (preferred arrangement set forth below)	6. Nucleotide and/or Amino Acid Sequence Submission (if applicable, all necessary)
- Descriptive title of the Invention	
- Cross References to Related Applications	
- Statement Regarding Fed sponsored R & D	
- Reference to Microfiche Appendix	
- Background of the Invention	
- Brief Summary of the Invention	
- Brief Description of the Drawings (if filed)	
- Detailed Description	
- Claim(s)	
- Abstract of the Disclosure	
3. <input checked="" type="checkbox"/> Drawing(s) (35 U.S.C. 113) [Total Sheets <input type="text" value="1"/>]	7. <input type="checkbox"/> Assignment Papers (cover sheet & document(s))
4. Oath or Declaration [Total Pages <input type="text" value="1"/>]	8. <input type="checkbox"/> 37 C.F.R. §3.73(b) Statement <input type="checkbox"/> Power of (when there is an assignee) <input type="checkbox"/> Attorney
a. <input checked="" type="checkbox"/> Newly executed (original or copy)	9. <input type="checkbox"/> English Translation Document (if applicable)
b. <input type="checkbox"/> Copy from a prior application (37 C.F.R. § 1.63(d)) (for continuation/divisional with Box 16 completed)	10. <input checked="" type="checkbox"/> Information Disclosure Statement (IDS)/PTO-1449 <input checked="" type="checkbox"/> Copies of IDS Citations
i. <input type="checkbox"/> DELETION OF INVENTOR(S) Signed statement attached deleting inventor(s) named in the prior application, see 37 C.F.R. §§ 1.63(d)(2) and 1.33(b).	
11. <input type="checkbox"/> Preliminary Amendment	
12. <input checked="" type="checkbox"/> Return Receipt Postcard (MPEP 503) (Should be specifically itemized)	
13. <input checked="" type="checkbox"/> * Small Entity Statement(s) <input type="checkbox"/> Statement filed in prior application, (PTO/SB/09-12) <input type="checkbox"/> Status still proper and desired	
14. <input type="checkbox"/> Certified Copy of Priority Document(s) (if foreign priority is claimed)	
15. <input type="checkbox"/> Other:	

* NOTE FOR ITEMS 1 & 13: IN ORDER TO BE ENTITLED TO PAY SMALL ENTITY FEES, A SMALL ENTITY STATEMENT IS REQUIRED (37 C.F.R. § 1.27), EXCEPT IF ONE FILED IN A PRIOR APPLICATION IS RELIED UPON (37 C.F.R. § 1.28).

16. If a CONTINUING APPLICATION, check appropriate box, and supply the requisite information below and in a preliminary amendment:

Continuation Divisional Continuation-in-part (CIP) of prior application No: _____ / _____

Prior application information: Examiner _____ Group / Art Unit: _____

For CONTINUATION or DIVISIONAL APPS only: The entire disclosure of the prior application, from which an oath or declaration is supplied under Box 4b, is considered a part of the disclosure of the accompanying continuation or divisional application and is hereby incorporated by reference. The incorporation can only be relied upon when a portion has been inadvertently omitted from the submitted application parts.

17. CORRESPONDENCE ADDRESS

<input type="checkbox"/> Customer Number or Bar Code Label	(Insert Customer No. or Attach bar code label here)			or <input checked="" type="checkbox"/> Correspondence address below
Name	Donald C. Erickson			
Address	Energy Concepts Co. 627 Ridgely Ave.			
City	Annapolis	State	MD	Zip Code
Country	USA	Telephone	410-266-6521	Fax
Name (Print/Type)		Donald C. Erickson		Registration No. (Attorney/Agent)
Signature		Donald C. Erickson		Date

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FEE TRANSMITTAL

for FY 1999

Patent fees are subject to annual revision.

Small Entity payments must be supported by a small entity statement, otherwise large entity fees must be paid. See Forms PTO/SB/09-12. See 37 C.F.R. §§ 1.27 and 1.28.

TOTAL AMOUNT OF PAYMENT (\$ 380

Complete if Known

Application Number			
Filing Date	SEPT 5, 2000		
First Named Inventor	Donald C. Erickson		
Examiner Name			
Group / Art Unit			
Attorney Docket No.	AIRCOMP		

09/654406 PRO 09/05/00

METHOD OF PAYMENT (check one)				FEE CALCULATION (continued)																																																																																																																															
<input checked="" type="checkbox"/> The Commissioner is hereby authorized to charge indicated fees and credit any over payments to:				3. ADDITIONAL FEES <table border="1"> <thead> <tr> <th>Large Entity Fee Code (\$)</th> <th>Small Entity Fee Code (\$)</th> <th>Fee Description</th> <th>Fee Paid</th> </tr> </thead> <tbody> <tr><td>105</td><td>130</td><td>205</td><td>65</td></tr> <tr><td>127</td><td>50</td><td>227</td><td>25</td></tr> <tr><td>139</td><td>130</td><td>139</td><td>130</td></tr> <tr><td>147</td><td>2,520</td><td>147</td><td>2,520</td></tr> <tr><td>112</td><td>920*</td><td>112</td><td>920*</td></tr> <tr><td>113</td><td>1,840*</td><td>113</td><td>1,840*</td></tr> <tr><td>115</td><td>110</td><td>215</td><td>55</td></tr> <tr><td>116</td><td>380</td><td>216</td><td>190</td></tr> <tr><td>117</td><td>870</td><td>217</td><td>435</td></tr> <tr><td>118</td><td>1,360</td><td>218</td><td>680</td></tr> <tr><td>128</td><td>1,850</td><td>228</td><td>925</td></tr> <tr><td>119</td><td>300</td><td>219</td><td>150</td></tr> <tr><td>120</td><td>300</td><td>220</td><td>150</td></tr> <tr><td>121</td><td>260</td><td>221</td><td>130</td></tr> <tr><td>138</td><td>1,510</td><td>138</td><td>1,510</td></tr> <tr><td>140</td><td>110</td><td>240</td><td>55</td></tr> <tr><td>141</td><td>1,210</td><td>241</td><td>605</td></tr> <tr><td>142</td><td>1,210</td><td>242</td><td>605</td></tr> <tr><td>143</td><td>430</td><td>243</td><td>215</td></tr> <tr><td>144</td><td>580</td><td>244</td><td>290</td></tr> <tr><td>122</td><td>130</td><td>122</td><td>130</td></tr> <tr><td>123</td><td>50</td><td>123</td><td>50</td></tr> <tr><td>126</td><td>240</td><td>126</td><td>240</td></tr> <tr><td>581</td><td>40</td><td>581</td><td>40</td></tr> <tr><td>146</td><td>760</td><td>246</td><td>380</td></tr> <tr><td>149</td><td>760</td><td>249</td><td>380</td></tr> <tr><td colspan="4">Other fee (specify)</td></tr> <tr><td colspan="4">Other fee (specify)</td></tr> <tr> <td colspan="4">* Reduced by Basic Filing Fee Paid</td> <td colspan="4">SUBTOTAL (3) (\$ 0</td> </tr> </tbody> </table>				Large Entity Fee Code (\$)	Small Entity Fee Code (\$)	Fee Description	Fee Paid	105	130	205	65	127	50	227	25	139	130	139	130	147	2,520	147	2,520	112	920*	112	920*	113	1,840*	113	1,840*	115	110	215	55	116	380	216	190	117	870	217	435	118	1,360	218	680	128	1,850	228	925	119	300	219	150	120	300	220	150	121	260	221	130	138	1,510	138	1,510	140	110	240	55	141	1,210	241	605	142	1,210	242	605	143	430	243	215	144	580	244	290	122	130	122	130	123	50	123	50	126	240	126	240	581	40	581	40	146	760	246	380	149	760	249	380	Other fee (specify)				Other fee (specify)				* Reduced by Basic Filing Fee Paid				SUBTOTAL (3) (\$ 0			
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Deposit Account Number 05-1067 Deposit Account Name Donald C. Erickson <input type="checkbox"/> Charge Any Additional Fee Required Under 37 CFR §§ 1.16 and 1.17				1. BASIC FILING FEE <table border="1"> <thead> <tr> <th>Large Entity Fee Code (\$)</th> <th>Small Entity Fee Code (\$)</th> <th>Fee Description</th> <th>Fee Paid</th> </tr> </thead> <tbody> <tr><td>101</td><td>760</td><td>201</td><td>380</td></tr> <tr><td>106</td><td>310</td><td>206</td><td>155</td></tr> <tr><td>107</td><td>480</td><td>207</td><td>240</td></tr> <tr><td>108</td><td>760</td><td>208</td><td>380</td></tr> <tr><td>114</td><td>150</td><td>214</td><td>75</td></tr> <tr><td colspan="4">SUBTOTAL (1) (\$ 380</td></tr> </tbody> </table> 2. EXTRA CLAIM FEES <table border="1"> <thead> <tr> <th>Total Claims</th> <th>Extra Claims</th> <th>Fee from below</th> <th>Fee Paid</th> </tr> </thead> <tbody> <tr><td>20</td><td>-20**</td><td>= 0</td><td>0</td></tr> <tr><td>3</td><td>- 3**</td><td>= 0</td><td>0</td></tr> <tr><td colspan="4">Multiple Dependent</td></tr> </tbody> </table> <p>**or number previously paid, if greater; For Reissues, see below</p> <table border="1"> <thead> <tr> <th>Large Entity Fee Code (\$)</th> <th>Small Entity Fee Code (\$)</th> <th>Fee Description</th> </tr> </thead> <tbody> <tr><td>103</td><td>18</td><td>203</td><td>9</td></tr> <tr><td>102</td><td>78</td><td>202</td><td>39</td></tr> <tr><td>104</td><td>260</td><td>204</td><td>130</td></tr> <tr><td>109</td><td>78</td><td>209</td><td>39</td></tr> <tr><td>110</td><td>18</td><td>210</td><td>9</td></tr> <tr><td colspan="4">** Reissue independent claims over original patent</td></tr> <tr><td colspan="4">** Reissue claims in excess of 20 and over original patent</td></tr> <tr><td colspan="4">SUBTOTAL (2) (\$ 0</td></tr> </tbody> </table>				Large Entity Fee Code (\$)	Small Entity Fee Code (\$)	Fee Description	Fee Paid	101	760	201	380	106	310	206	155	107	480	207	240	108	760	208	380	114	150	214	75	SUBTOTAL (1) (\$ 380				Total Claims	Extra Claims	Fee from below	Fee Paid	20	-20**	= 0	0	3	- 3**	= 0	0	Multiple Dependent				Large Entity Fee Code (\$)	Small Entity Fee Code (\$)	Fee Description	103	18	203	9	102	78	202	39	104	260	204	130	109	78	209	39	110	18	210	9	** Reissue independent claims over original patent				** Reissue claims in excess of 20 and over original patent				SUBTOTAL (2) (\$ 0																																																
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SUBMITTED BY		Complete (if applicable)		
Name (Print/Type)	Donald C. Erickson	Registration No. (Attorney/Agent)		Telephone 410-266-6521
Signature	Donald C Erickson			Date 9/5/00

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STATEMENT CLAIMING SMALL ENTITY STATUS (37 CFR 1.9(f) & 1.27(b))--INDEPENDENT INVENTOR	Docket Number (Optional) AIRCOMP
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Applicant, Patentee, or Identifier: DONALD C. ERICKSON

Application or Patent No.: _____

Filed or Issued: SEPTEMBER 5, 2000Title: AIR COMPRESSION IMPROVEMENT

As a below named inventor, I hereby state that I qualify as an independent inventor as defined in 37 CFR 1.9(c) for purposes of paying reduced fees to the Patent and Trademark Office described in:

the specification filed herewith with title as listed above.

the application identified above.

the patent identified above.

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Each person, concern, or organization to which I have assigned, granted, conveyed, or licensed or am under an obligation under contract or law to assign, grant, convey, or license any rights in the invention is listed below:

No such person, concern, or organization exists.

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Separate statements are required from each named person, concern, or organization having rights to the invention stating their status as small entities. (37 CFR 1.27)

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<u>DONALD C. ERICKSON</u>	NAME OF INVENTOR	NAME OF INVENTOR
<u>Ronald C. Erickson</u>	Signature of inventor	Signature of inventor
<u>9/5/00</u>	Date	Date

Certification under 37 CFR 1.10 (if applicable)

EE451582748US

"Express Mail" mailing number

915700

Date of Deposit

I hereby certify that this application is being deposited with the United States Postal Service "Express Mail Post Office to Addressee" service under 37 CFR 1.10 on the date indicated above and is addressed to the Commissioner of Patents and Trademarks, Washington, D.C. 20231.

MARY WITTNER

(Typed or printed name of person
mailing application)

Mary Miller

(Signature of person mailing application)

AIR COMPRESSION IMPROVEMENT

CROSS-REFERENCE TO RELATED APPLICATIONS

Not applicable.

5

STATEMENT REGARDING THE FEDERALLY SPONSORED RESEARCH OR DEVELOPMENT

Not applicable.

10 REFERENCE TO A MICROFICHE APPENDIX

Not applicable.

BACKGROUND OF THE INVENTION

All types of air compressors share an ambient temperature sensitivity - both the

15 capacity and the efficiency decrease as the ambient temperature increases. The compressor-specific power demand is approximately proportional to the absolute temperature, which makes the efficiency proportional to the inverse absolute temperature. The compressor capacity is proportional to the density of the inlet air.

These sensitivities become particularly pronounced in combustion engines, in which

20 the compressed air is used to combust a fuel and ultimately produce power. Both the power output and engine efficiency are de-rated at warm ambients. The degradation is not so severe with reciprocating engines, which require little more than stoichiometric air. The degradation is very severe with combustion turbines, which require on the order of 3 or 4 times stoichiometric air.

25 One known method of counteracting the warm ambient degradation of air compressors is by cooling the inlet air, either evaporatively or with a refrigerant. The refrigerated cooling can be done either in refrigerated air coils or by direct contact with sprayed chilled water. The refrigeration is supplied by either mechanical or absorption refrigeration systems, and in some instances through a cold storage medium (ice or chilled

30 water).

Another approach to cooling inlet air is by over-spraying, typically via fogging.

Sufficient water is injected into the air in fine droplet form such that it not only reduces the temperature adiabatically to the dew point, but additional droplets remain un-evaporated,

GOVERNMENT EDITION

and carry into the compressor suction. Those droplets rapidly evaporate as compression proceeds, slowing the temperature increase caused by compression, and hence effectively adding to the amount of inlet cooling. For the droplets to remain suspended in the air into the suction rather than separate out excessively, they should be in the fog-size range, i. e.,

5 less than 40 microns in diameter and preferably 5 to 20 microns. Another advantage of this size range is that the droplets are small enough that they do not erode the compressor blades.

The problems with the current approaches to cooling compressor inlet air include the following. Most compressors would benefit thermodynamically from sub-freezing inlet

10 temperatures, or at least could be designed to benefit from those temperatures. However, there are many practical difficulties. Especially with high rotational speed combustion turbines, there is a possibility of ice buildup on inlet guide vanes, which then could spall off and damage the compressor blades. This imposes a practical limiting temperature of about 4°C for many inlet cooling systems. Cooling below that temperature will require some 15 additional technique of reducing the humidity level of the cold air below saturation - reheat, etc. On the refrigeration side, special measures are also required to deal with the H₂O removal from the air in sub-freezing conditions: periodic defrosting of the air coils, or continuous addition of a melting agent. Furthermore, the refrigeration system requires proportionately more input power to reach the lower temperatures - more shaft power for 20 mechanical refrigeration, or higher quality heat for absorption refrigeration. With mechanical refrigeration, the power necessary to reach sub-freezing temperatures is so large, and the marginal improvement in compression is so small, that there is little or no net gain from cooling to sub-freezing temperatures.

Even when the inlet cooling is restricted to above-freezing temperatures, another

25 major problem remains. The compressor benefit is substantially due to the sensible cooling of the inlet air, with almost no added benefit from the latent cooling, i.e., the amount of moisture condensed out of the air. However, the latent cooling typically represents 25 to 50% of the total refrigeration load. For example, consider 35°C air at 50% relative humidity, which is cooled to 5°C at 100% relative humidity. The moisture content decreased from 1.8 30 weight percent to 0.55 weight percent. For these conditions, only 51% of the total refrigeration provides sensible cooling, and 49% causes the water condensation. Thus, much of the refrigeration is effectively wasted.

The overspray or fogging approach to inlet cooling also presents problems. The two foremost are that the cooling is adiabatic, as opposed to the diabatic cooling of the refrigeration approach; and that a source of pure water is required for every bit of cooling accomplished. The adiabatic limitation causes the inlet sensible temperature to be no lower than the dew point. The cost and availability of pure water mitigate against this approach at many sites.

What is needed, and included among the objects of this invention, are apparatus and process which overcome the prior art problems cited above, i. e., an inlet cooling system wherein the latent load contributes to effective cooling in addition to the sensible load contribution; where the benefits of the overspray approach are available without the limitations of needing a large source of pure water and that the inlet temperature is limited to the dew point; where the thermodynamic benefits of sub-freezing inlet temperatures are achievable without the practical problems; and wherein the refrigeration system is activated by low temperature waste heat so as not to detract from the compressor shaft power reduction provided by the inlet cooling system.

DISCLOSURE OF THE INVENTION

The above advantages are obtained in a process for compressing air comprising: chilling air to between the dew point and the frost point; collecting the resulting condensate; injecting the condensate into the chilled air in the form of very small droplets; and compressing the chilled droplet laden air. They are also obtained in an apparatus for increasing the capacity and efficiency of an air compressor comprising: a means for air chilling which is supplied with a refrigerant; a condensate collection system for condensate condensed from said air by said means for chilling; a means for converting said condensate into fog-sized droplets; a means for injecting said droplets into said air downstream of said chilling means; and a duct for supplying said chilled and fogged air to the suction of said air compressor.

BRIEF DESCRIPTION OF THE SEVERAL VIEWS OF THE DRAWINGS

Figure 1 illustrates the three essential features of the invention: an air chilling system including means for condensate collection; an overspray system; and an air compressor.

Figure 2 shows a more complex application of the invention wherein the compressor is part of a combustion engine, and the engine waste heat powers an absorption refrigeration unit (ARU) which in turn supplies chilling to the air chiller.

5 DETAILED DESCRIPTION OF THE INVENTION

Referring to Figure 1, inlet air for air compressor 10 is first supplied to air chiller 11, where it is cooled to below the dew point by cooling coil 12. The condensate is collected in collection pan 13, then pressurized to between 6 and 20 MPa in pump 14, and routed to fogging nozzles 15 of overspray system 16. From there, the chilled, saturated, oversprayed air is routed to the suction of compressor 10. There may also be a spray water reservoir, filter, makeup source, and deionizing bed, to help ensure continuity and purity of the spray.

Referring to Figure 2, combustion turbine 20 is comprised of compressor 21, turbine 22, combustor 23, and regenerator 24. Inlet air for compressor 21 is filtered in filter 25, chilled to below the dew point in refrigerated air coil 26, and then fogged by spray nozzles 27. Liquid refrigerant is supplied to air coil 26 from ARU 28 via pressure letdown valve 29, and refrigerant vapor is returned to the ARU. Moisture condensed from the air is collected in collector 30, filtered and purified in filter-purifier 31, and pressurized in pump 32, then routed to the fogging nozzles 27. The heat of compression in compressor 21 evaporates all the fog droplets, and compressed air exits the compressor with the benefits of both sensible and latent cooling, and at a correspondingly lower temperature. The maximum thermodynamic benefit is obtained when the cooler compressed air is supplied to regenerator 24, as shown, although substantial benefit is also obtained without a regenerator. Fuel 33 is combusted with the compressed air in combustor 23, and the hot pressurized combustion gas is expanded in turbine 22 to produce shaft power. The hot exhaust may be routed through regenerator 24, diverter valve 34, heat recovery steam generator 35, and finally ARU 28, before exhausting to atmosphere through stack 36.

With the Figure 2 flowsheet, and assuming the operating conditions cited above (35°C, 50% relative humidity ambient, chilled to 5°C) the following benefits are achieved. The inlet air is sensibly cooled by 30°C, plus additional overspray cooling internal to the compressor of virtually the same amount (60°C cooling altogether). The turbine shaft power output increases by at least about 30%, and the efficiency increases by 5 to 20%, dependent upon the pressure ratio and whether or not regeneration is present. The maximum efficiency increase is obtained with regeneration, and with the lower pressure

ratio machines such as microturbines. Even with large combined cycle plants, an appreciable overall plant efficiency gain is realized, in addition to the major gain in capacity. The large amount of effective inlet cooling is achieved without the problems of sub-freezing conditions, and without need for a separate source of pure water for the fogging system.

5 Since waste heat powers the absorption system there is almost no parasitic power offset to the increased capability.

The NH₃ - H₂O type of ARU adapts well to being directly heated by low temperature exhaust, e.g., 175°C or lower, and also to direct expansion chilling coils. However, LiBr ARUs may also be used, and need not be directly integrated, i.e., can use steam or hot 10 water heating and chill water cooling circuit. The air cooling to below the dew point can be via direct contact, e.g., with a spray of recirculating chilled water, rather than via coils. With coils, more than one evaporation temperature can profitably be used.

The NH₃ - H₂O ARU can also be used to make ice, e.g., for thermal storage cooling of a peaking or variably loaded plant. With a combustion engine, the 60°C cooling cited above can be driven by as little as 100°C cooling of the exhaust, e.g., from 175°C to 75°C. For some applications it will be desirable to further refrigerate the inlet air to below freezing before fogging, and/or to do interstage fogging in lieu of inlet fogging. Compressed air supply systems will also benefit from this disclosure, plus also other types of combustion engines, such as reciprocating types.

20 Standard means of generating fog-sized droplets are contemplated, including the techniques described in the enclosed references. The refrigeration for chilling can be from mechanical compression systems in lieu of by absorption.

CLAIMS

1. A process for compressing air comprising:

- chilling air to between the dew point and the frost point;
- 5 • collecting the resulting condensate;
- injecting the condensate into the chilled air in the form of very small droplets; and
- compressing the chilled droplet laden air.

2. The process according to claim 1 wherein said droplets are predominantly in the size

10 range of 5 to 40 microns - normally referred to as fog.

3. The process according to claim 2 wherein said chilling is to a temperature below about
5°C.

15 4. The process according to claim 2 additionally comprising combusting a fuel with said
compressed air; and work expanding the resulting hot compressed combustion products.

5. The process according to claim 2 additionally comprising supplying said chilling by an
absorption refrigeration unit (ARU).

20 6. The process according to claim 5 additionally comprising combusting a fuel with said air
and work expanding the resulting hot combustion products; and supplying heat to said ARU
from said work expander exhaust.

25 7. The process according to claim 6 wherein said ARU is an ammonia-absorption type, and
additionally comprising supplying ARU ammonia refrigerant directly to an air coil for said
chilling step; and providing exhaust heating directly to the ARU absorbent.

30 8. The process according to claim 2 additionally comprising partially compressing said
chilled air prior to injecting said fog droplets.

9. The process according to claim 2 additionally comprising refrigerating said chilled air to
below the frost point before injecting fog.

10. An apparatus for increasing the capacity and efficiency of an air compressor comprising:

- a means for air chilling which is supplied with a refrigerant;
- a condensate collection system for condensate condensed from said air by said means for chilling;
- a means for converting said condensate into fog-sized droplets;
- a means for injecting said droplets into said air downstream of said chilling means; and
- a duct for supplying said chilled and fogged air to the suction of said air compressor.

11. The apparatus according to claim 10 wherein said means for air chilling is comprised of refrigerated air coils.

12. The apparatus according to claim 11 additionally comprised of an ARU which supplies refrigerant directly to said air coils.

13. The apparatus according to claim 12 wherein said ARU is comprised of $\text{NH}_3 - \text{H}_2\text{O}$ working fluid, and a heat exchanger between said working fluid and a combustion exhaust gas.

14. The apparatus according to claim 13 wherein said combustion exhaust gas is from a combustion engine which is supplied by said air compressor.

15. The apparatus according to claim 14 wherein said combustion engine is a reciprocating engine.

16. The apparatus according to claim 14 wherein said combustion engine is a combustion turbine.

17. The apparatus according to claim 16 wherein said combustion turbine includes a regenerator.

18. The apparatus according to claim 10 additionally comprised of a LiBr ARU which supplies said chilling.

5 19. An apparatus for increasing the efficiency of a combustion turbine comprising:

- a) a chiller for the inlet air for the combustion turbine which chills said air to below the dew point;
- b) a collector for condensate from said chiller; and
- c) a system for injecting said condensate into said chilled air in the form of fog-sized

10 droplets.

20. The apparatus according to claim 19 additionally comprised of an ARU which supplies cooling to said chiller and which is supplied waste heat from said combustion turbine exhaust; and at least one of:

15 a) a heat recovery steam generator; and

- b) a regenerator.

ABSTRACT OF THE DISCLOSURE

The efficiency and capacity of an air compressor (10) (Figure 1) are increased by pre-cooling the inlet air to below the dew point in air chiller (11), and then injecting the resulting condensate into the chilled air in the form of fog-sized droplets in a fogger (16).
5 The advantages extend to combustion engines, and especially to regenerative combustion turbines.

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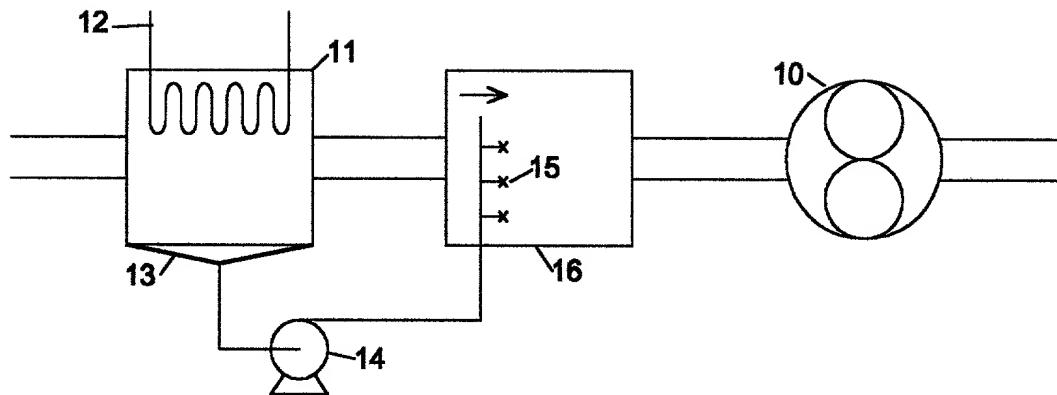


Figure 1

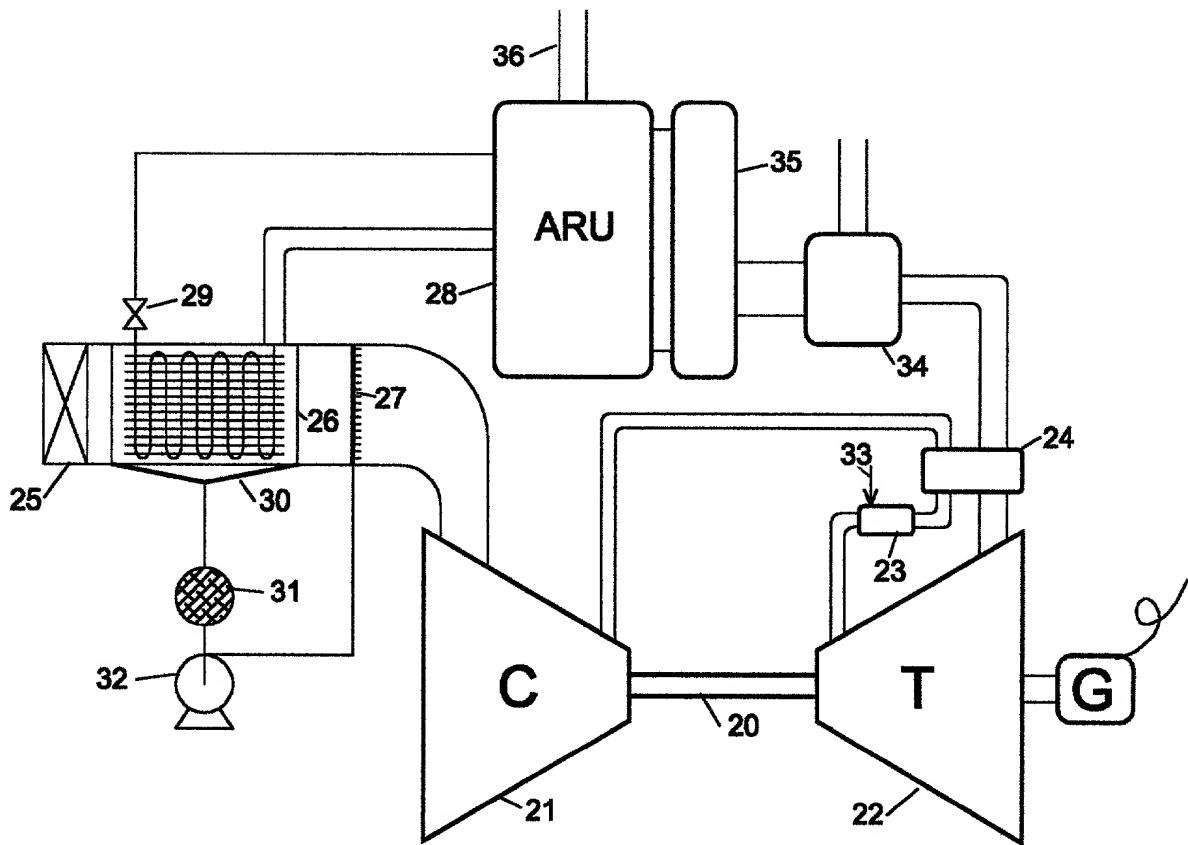


Figure 2

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PTO/SB/01 (12-97)

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**DECLARATION FOR UTILITY OR
DESIGN
PATENT APPLICATION
(37 CFR 1.63)**

Declaration
Submitted with Initial
Filing Declaration
Submitted after Initial
Filing (surcharge
(37 CFR 1.16 (e))
required)

Attorney Docket Number	AIRCOMP
First Named Inventor	Donald C. Erickson
COMPLETE IF KNOWN	
Application Number	/
Filing Date	SEPT. 5, 2000
Group Art Unit	
Examiner Name	

As a below named Inventor, I hereby declare that:

My residence, post office address, and citizenship are as stated below next to my name.

I believe I am the original, first and sole inventor (if only one name is listed below) or an original, first and joint inventor (if plural names are listed below) of the subject matter which is claimed and for which a patent is sought on the invention entitled:

AIR COMPRESSION IMPROVEMENT

the specification of which

(Title of the Invention)

is attached hereto

OR

was filed on (MM/DD/YYYY)

as United States Application Number or PCT International

Application Number and was amended on (MM/DD/YYYY) (if applicable).

I hereby state that I have reviewed and understand the contents of the above identified specification, including the claims, as amended by any amendment specifically referred to above.

I acknowledge the duty to disclose information which is material to patentability as defined in 37 CFR 1.56.

I hereby claim foreign priority benefits under 35 U.S.C. 119(a)-(d) or 365(b) of any foreign application(s) for patent or inventor's certificate, or 365(a) of any PCT international application which designated at least one country other than the United States of America, listed below and have also identified below, by checking the box, any foreign application for patent or inventor's certificate, or of any PCT international application having a filing date before that of the application on which priority is claimed.

Prior Foreign Application Number(s)	Country	Foreign Filing Date (MM/DD/YYYY)	Priority Not Claimed	Certified Copy Attached?
			<input type="checkbox"/>	<input type="checkbox"/>
			<input type="checkbox"/>	<input type="checkbox"/>
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			<input type="checkbox"/>	<input type="checkbox"/>

Additional foreign application numbers are listed on a supplemental priority data sheet PTO/SB/02B attached hereto.

I hereby claim the benefit under 35 U.S.C. 119(e) of any United States provisional application(s) listed below.

Application Number(s)	Filing Date (MM/DD/YYYY)	
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[Page 1 of 2]

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Additional registered practitioner(s) named on supplemental Registered Practitioner Information sheet PTO/SB/02C attached hereto.

Direct all correspondence to: Customer Number OR Correspondence address below

Name	Donald C. Erickson				
Address	Energy Concepts Company				
Address	627 Ridgely Ave.				
City	Annapolis	State	MD	ZIP	21401
Country	USA	Telephone	410-266-6521		Fax 410-266-6539

I hereby declare that all statements made herein of my own knowledge are true and that all statements made on information and belief are believed to be true; and further that these statements were made with the knowledge that willful false statements and the like so made are punishable by fine or imprisonment, or both, under 18 U.S.C. 1001 and that such willful false statements may jeopardize the validity of the application or any patent issued thereon.

Name of Sole or First Inventor: A petition has been filed for this unsigned inventor

Given Name (first and middle if any) Family Name or Surname

Donald C. Erickson

Inventor's Signature	Donald C. Erickson				Date	9/5/00
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Residence: City	Annapolis	State	MD	Country	USA	Citizenship	US
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Post Office Address	1704 South Harbor Lane						
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Additional inventors are being named on the _____ supplemental Additional Inventor(s) sheet(s) PTO/SB/02A attached hereto